

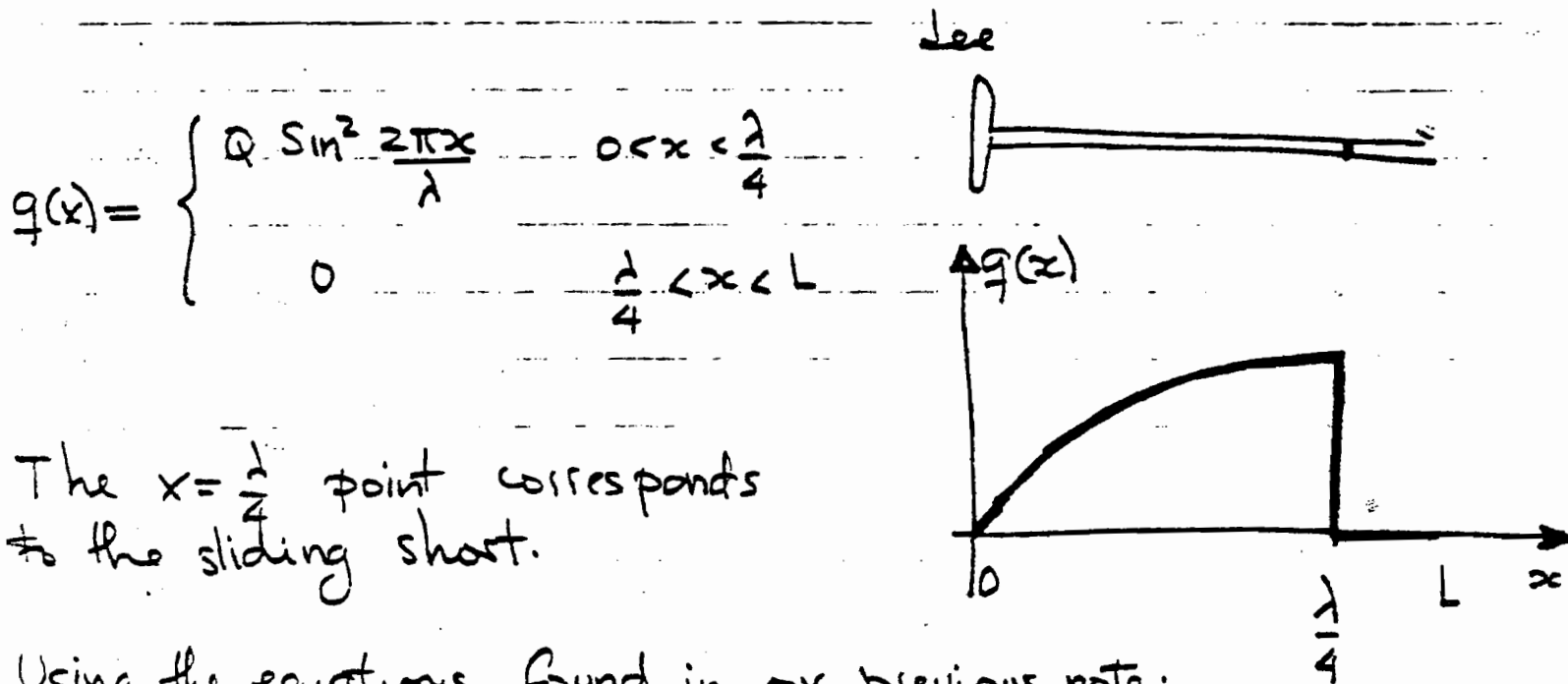
Felix Marti

1/22/80 (1)

second Approximation to the problem of.
Heat transfer in the dee stem.

We had previously assumed that the power dissipated per unit length on the inner cylinder of the dee stem was constant

We will consider now a more realistic distribution



The $x = \frac{\lambda}{4}$ point corresponds to the sliding short.

Using the equations found in our previous note:

$$\frac{d^2 T_1}{dx^2} = -\alpha(x) = -\frac{\ell h}{(\eta c_p)^2 \left(2 + \frac{e_1 h}{k}\right)} q(x)$$

$$\frac{d^2 T_1}{dx^2} = -\tau \sin^2\left(\frac{2\pi x}{\lambda}\right)$$

with $\tau = \frac{\ell h Q}{(\eta c_p)^2 \left(2 + \frac{e_1 h}{k}\right)}$

integrating:

(2)

$$T_1(x) = -\tau \left[\frac{x^2}{4} + \frac{\lambda^2}{32\pi^2} \cos\left(\frac{4\pi x}{\lambda}\right) \right] + \beta'x + \gamma'$$

$$T_2(x) = T_1(x) + \frac{\dot{m} c_p}{\ell h} \left(2 + \frac{e_1 h}{k} \right) \frac{dT_1}{dx}$$

Boundary conditions: $T_2(0) = T_1(0) \Rightarrow \frac{dT_1}{dx}(0) = \beta' = 0$

For $\frac{\lambda}{4} < x < L$ $q = 0$ $\frac{d^2 T_1}{dx^2} = 0$

$$T_1 = q x + c_2$$

$$T_2 = q x + c_2 + \frac{\dot{m} c_p}{\ell h} \left(2 + \frac{e_1 h}{k} \right) q$$

$T_2(L) = T_0 = \text{temperature entrance water. } \approx 20^\circ.$

Solving for γ'

$$\gamma' = -\tau \frac{\lambda^2}{32} + T_0 + \frac{Q \lambda / 8}{\dot{m} c_p} + \tau \frac{\lambda}{8} L + \frac{\tau \lambda^2}{32} \left(\frac{1}{2} - \frac{1}{\pi^2} \right)$$

$$T_{\max} = \gamma' - \frac{\tau \lambda^2}{32\pi^2} = T_0 + \frac{Q \lambda / 8}{\dot{m} c_p} + \tau \frac{\lambda}{8} L - \frac{\tau \lambda^2}{32} \left(\frac{1}{2} + \frac{2}{\pi^2} \right)$$

(3)

The total power dissipated is

$$W = \int_0^{\lambda/4} Q \sin^2\left(\frac{2\pi x}{\lambda}\right) dx = Q \left(\frac{\lambda}{8}\right)$$

Case (a) $\frac{\lambda}{4} = 17$ ft. (low frequency) $W = 33$ kW

Using the values found in the 1st approximation study we obtain

$$T_{\max} = T_0 + 50 \quad ^\circ\text{C}$$

$$T_1(x) = 70 - \frac{x^2}{10} + 5.9 \left[1 - \cos\left(\frac{x}{5.41}\right) \right]$$

Fig 1 shows the water temperature as a function of x .

Case (b) $\frac{\lambda}{4} = 5$ ft (high frequency) $W = 33$ kW

$$T_{\max} = T_0 + 78 \quad ^\circ\text{C}$$

$$T_1(x) = 98 - .34x^2 + 1.73 \left(1 - \cos\frac{x}{1.6} \right)$$

for $0 \leq x < \frac{\lambda}{4}$ and changes linearly with x for

$\frac{\lambda}{4} < x < L$. Fig 2 shows T vs. x .

(4)

After we made these graphs new information indicated that the maximum power was 20 kW and not 33 as we had assumed, and that the water flow was close to 3.4 gallons per minute and not 4.

Taking into account possible changes in these parameters the maximum temperature can be found to be given by:

$$T_{\max} = T_0 + 3.9 \frac{W}{g} + 22.3 \frac{W}{g^2}$$

Where W = total power dissipated on inner conductor of Lee stem.

g = gallons per minute of water.

The worst case occurs for the high frequency case.

For $W = 20 \text{ kW}$ and $g = 3.4 \text{ gpm}$ $T_{\max} = T_0 + 62^\circ\text{C}$
with $T_0 = 20^\circ\text{C}$

$$T_{\max} \approx 82^\circ\text{C}$$

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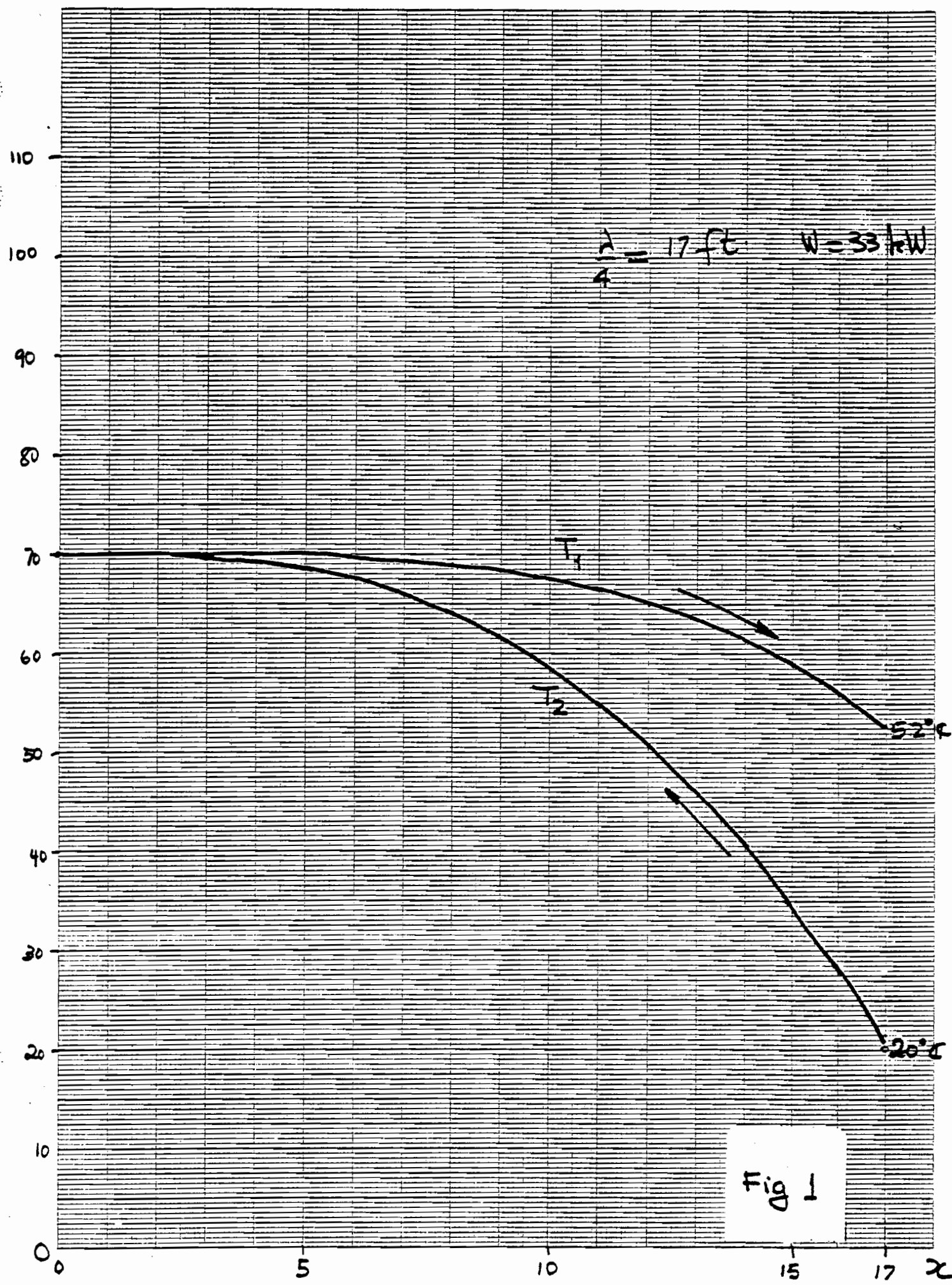


Fig 1

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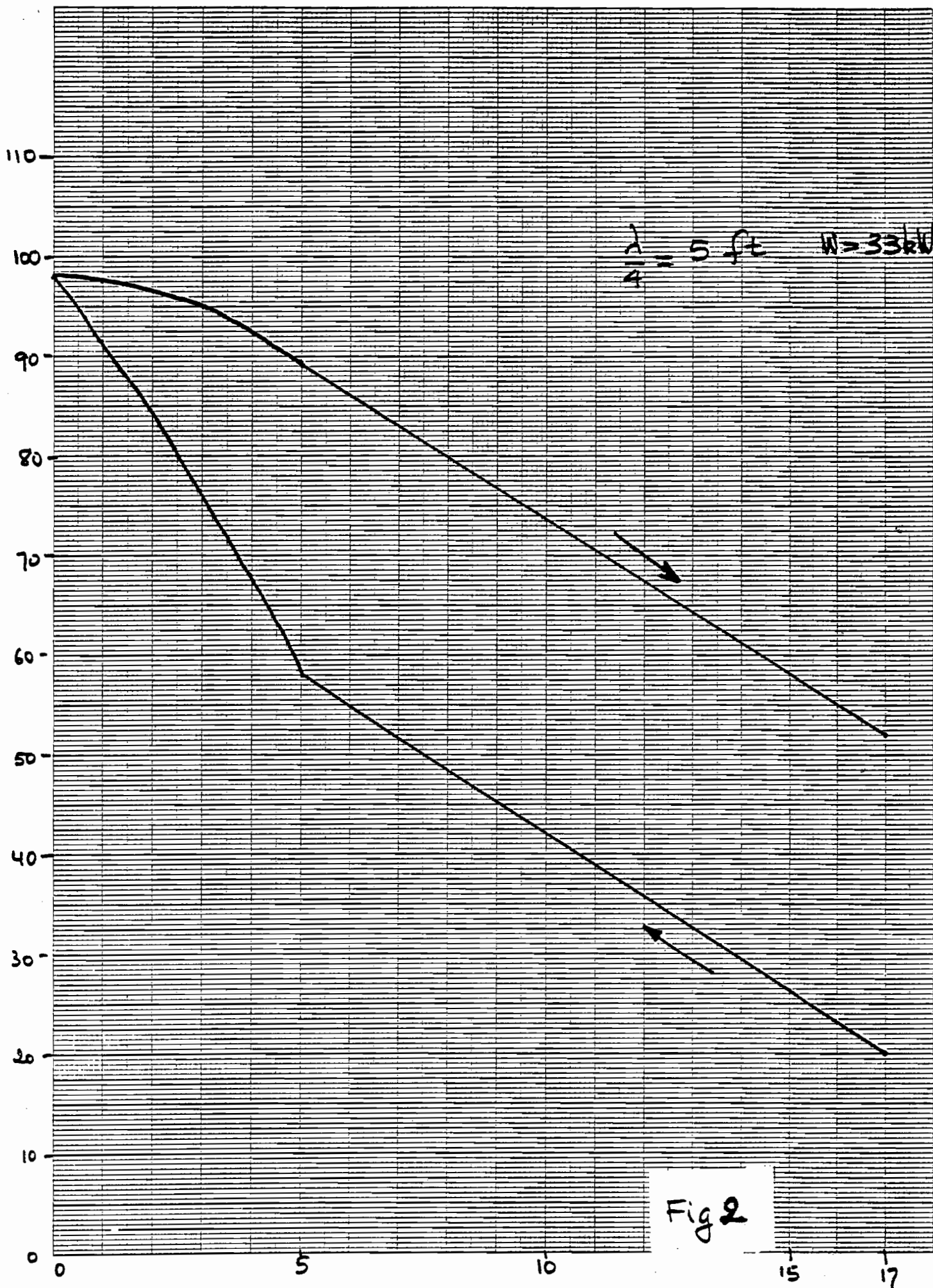
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Fig 2